

|  | INSTITUTE OF AERONAUTICAL ENGINEERING |
| :---: | :---: |
| (Autonomous) |  |
| Dundigal, Hyderabad - 500 043 |  |

Time: 3 hour
Maximum Marks: 70

## Answer ONE Question from each MODULE All Questions Carry Equal Marks <br> All parts of the question must be answered in one place only MODULE-I

1. (a) Explain the angle of heel in the case of a two wheeler negotiating a curve.
[7m]
(b) The mass of the motor cycle along with the rider is 180 kg . The height of the centre of gravity of total mass is 600 mm above the ground when it moves straight. Each wheel has a diameter of 700 mm and mass moment of inertia of 2 kgm The engine rotates at a speed of 5 times the road wheel 2and engine rotating parts have mass moment of inertia of 0.2 kgm . Find the angle of heel if it is travelling at $50 \mathrm{~km} / \mathrm{h}$ and is taking a turn of 30 meters radius.
[7m]
2. (a) Explain the effect of gyroscopic couple on an automobile taking left turn.
[7m]
(b) A uniform disc having a mass of 8 kg and radius of gyration 150 mm is mounted on one end of a horizontal arm of length 200 mm . The other end rotates freely in a bearing. The disc is given a clockwise spin of 240 rpm . Determine the motion of the disc if its arm remains horizontal.
[7m]

## MODULE-II

3. (a) Deduce expression for the friction torque for a flat collar bearing considering uniform wear. [7m]
(b) A conical pivot supports a load of $10 k N$, cone angle is 1100 and intensity of pressure normal to the cone is $0.3 \mathrm{~N} / \mathrm{mm}^{2}$. The outer diameter is twice the inner diameter. Find the outer and inner radii of bearing surface if the shaft rotates at 300 rpm and $\mu=0.1$. Find the power absorbed in friction assuming uniform wear.
4. (a) Deduce expression for the friction torque for a flat collar bearing considering uniform pressure.
[7m]
(b) A band brake acts on 3/4th of a circumference of a brake drum of 450 mm diameter which is keyed to a shaft. The band brake provides a braking torque of 225 Nm . One end of the
lever is attached to a fulcrum pin of the lever and the other end is attached to a pin 100 mm from the fulcrum. If the operating force is applied at 500 mm from the fulcrum and coefficient of friction is 0.25 , find the operating force when the drum rotates in Clock-wise direction and anti- clockwise direction.
[7m]

## MODULE-III

5. (a) Describe the graphical method of determining the inertia of the connecting rod of reciprocating engine.
[7m]
(b) The turning moment diagram for a multi cylinder engine has been drawn to a scale of 1 mm to 500 Nm of torque and 1 mm to 60 of crank displacement The intercepted areas between the output torque curve and the mean resistance line taken in order from one end of the engine are $-30,+410,-280,+320,-330,+250,-360,+280,-260 \mathrm{~mm}^{2}$ when the engine runs at 800 rpm . The engine has a stroke of 300 mm and the fluctuation of speed is not to exceed $2 \%$ of mean speed. Determine suitable diameter and cross section of the flywheel rim for a limiting value of safe centrifugal stress of $7 M P$. The material density is $720 \mathrm{~kg} / \mathrm{m}^{3}$. Width of the rim is 5 times the thickness.
[7m]
6. (a) Derive an expression for the angular acceleration of the connecting rod of a reciprocating engine.
[7m]
(b) The effective turning moment exerted by a two stroke engine at the crank shaft is $T=$ $800+100 \sin 2 \theta-200 \cos \theta$ where $\theta$ is the inclination of the crank to inner dead center. The mass of the flywheel is 400 kg and radius of gyration is 550 mm . The engine speed is 200 rpm . Determine the power developed, the total percentage fluctuation of speed and maximum angular retardation.
[7m]

## MODULE-IV

7. (a) Explain the balancing of reciprocating masses with a neat sketch.
[7m]
(b) A rigid rotor has its unbalance in one plane and can be considered to consist of three masses $m_{1}=5 \mathrm{~kg}$ at an angle of 300 from mass m 1 in anti clockwise direction, $m_{2}=3 \mathrm{~kg}$ at an angle of 1650 counterclockwise from $m_{1}$ and $m_{3}=8 \mathrm{~kg}$ at angle $85^{\circ}$ clockwise from $m_{1}$. The radii $r_{1}=200 \mathrm{~mm}, r_{2}=80 \mathrm{~mm}$ and $r_{3}=140 \mathrm{~mm}$. Determine the balancing mass required at a radius of 100 mm . Specify the location of this mass with respect to $m_{1} \cdot[\mathbf{7 m}]$
8. (a) Derive the expression for hammer blow in locomotive balancing.
[7m]
(b) The cranks of a three cylinder locomotive are set at 1000 . The stroke is 110 mm , the length of the connecting rod is 140 mm , the mass of the reciprocating parts per cylinder is 1 Kg and the speed of the crank shaft is 2400 rpm . Determine the magnitude of primary and secondary balancing.
[7m]

## MODULE-V

9. (a) Derive an expression for the natural frequency of forced longitudinal vibrations. [7m]
(b) A shaft 100 mm diameter and 1000 mm long is fixed at one end and the other end carries a flywheel of mass 90 kg . The radius of gyration of the flywheel is 500 mm . Find the frequency of torsional vibration, if the modulus of rigidity for the shaft material is $80 G N / m^{2}$.
10. (a) What is meant by the term critical damping?
(b) A cantilever shaft of 50 mm diameter and 300 mm long has a disc of mass 100 kg at its free end. The Youngs modulus of the shaft material is $200 \mathrm{GN} / \mathrm{m}^{2}$. Determine the frequency of longitudinal and transverse vibrations of the shaft.

## **END OF EXAMINATION**

## COURSE OBJECTIVES:

The course should enable the students to:

| 1 | The concepts of precision, static and dynamic forces of planer mechanisms by <br> neglecting friction of aero planes, sea vessels, auto mobiles and various force members. |
| :---: | :--- |
| 2 | The knowledge of engineering mechanics for identifying the coefficient of friction and <br> engine speed of the various contact bodies (Clutches and Brakes) and speed controlled <br> devices, variations of torques and fluctuation of speeds of IC engines. |
| 3 | The magnitude and direction of balanced mass for unbalanced rotary and <br> reciprocating engines with the fundamentals of applied physics. |
| 4 | Mathematical modeling of various degree of freedom systems to interpret the various <br> vibration parameters. |
| 5 | The affluence of real world engineering problems and examples towards gaining the <br> experience for how dynamics of machinery is applied in engineering practice. |

## COURSE OUTCOMES:

After successful completion of the course, students should be able to:

| CO 1 | Discuss the Gyroscopes, effect of precession motion on the stability of moving vehicles <br> such as motor car, motor cycle, aeroplanes and ships. |
| :---: | :--- |
| CO 2 | Determine the angle of heel to avoid upside down of a two wheeler vehicle while <br> taking in left and right turns. |
| CO 3 | Illustrate the static and dynamic force analysis of two and three force members by <br> graphical super position method. |
| CO 4 | Apply the laws of friction on clutches, brakes and dynamometers to reduce the power <br> losses for the effective torque transmission. |
| CO 5 | Justify the importance of torque and fluctuation of speeds for single and multi <br> cylindered engines to increase the mechanical efficiency. |
| CO 6 | Estimate the height of a governor to regulate the speed of a prime mover at various <br> load conditions. |
| CO 7 | Determine the balanced mass for unbalanced rotary and reciprocating engines by <br> analytical and graphical methods. |
| CO 8 | Develop a mathematical modelling of free and forced vibration systems under damped <br> and un-damped conditions to avoid the vibratory damages of aero-mechanical-civil <br> structures and electrical and electronic components at various operated frequencies. |
| CO 9 10 | Use the resonance phenomenon to predict the critical or whirling or whipping speeds <br> of various structures under vibrations to avoid catastrophic failures. |
| Apply the principles of dynamics of machinery to a real world problems for obtaining <br> optimum solutions. |  |

MAPPING OF SEMESTER END EXAMINATION QUESTIONS TO COURSE OUTCOMES

| Q.No |  | All Questions carry equal marks | Taxonomy | CO's | PO's |
| :---: | :---: | :---: | :---: | :---: | :---: |
| 1 | a | Explain the angle of heel in the case of a two wheeler negotiating a curve. | Understand | CO 1 | PO 1 |
|  | b | The mass of the motor cycle along with the rider is 180 kg . The height of the centre of gravity of total mass is 600 mm above the ground when it moves straight. Each wheel has a diameter of 700 mm and mass moment of inertia of 2 kgm The engine rotates at a speed of 5 times the road wheel 2and engine rotating parts have mass moment of inertia of 0.2 kgm . Find the angle of heel if it is travelling at $50 \mathrm{~km} / \mathrm{h}$ and is taking a turn of 30 meters radius. | Apply | CO 2 | PO 2 |
| 2 | a | Explain the effect of gyroscopic couple on an automobile taking left turn. | Understand | CO 2 | PO 1,2 |
|  | b | A uniform disc having a mass of 8 kg and radius of gyration 150 mm is mounted on one end of a horizontal arm of length 200 mm . The other end rotates freely in a bearing. The disc is given a clockwise spin of 240 rpm . Determine the motion of the disc if its arm remains horizontal. | Apply | CO 1 | PO 1 |
| 3 | a | Deduce expression for the friction torque for a flat collar bearing considering uniform wear. | Understand | CO 4 | PO 1 |
|  | b | A conical pivot supports a load of 10 kN , cone angle is 1100 and intensity of pressure normal to the cone is $0.3 \mathrm{~N} / \mathrm{mm}_{2}$. The outer diameter is twice the inner diameter. Find the outer and inner radii of bearing surface if the shaft rotates at 300 rpm and $\mu=0.1$. Find the power absorbed in friction assuming uniform wear. | Apply | CO 4 | PO 1 |
| 4 | a | Deduce expression for the friction torque for a flat collar bearing considering uniform pressure. | Understand | CO 4 | PO 1 |


|  | b | A band brake acts on $3 / 4$ th of a circumference of a brake drum of 450 mm diameter which is keyed to a shaft. The band brake provides a braking torque of 225 Nm . One end of the lever is attached to a fulcrum pin of the lever and the other end is attached to a pin 100 mm from the fulcrum. If the operating force is applied at 500 mm from the fulcrum and coefficient of friction is 0.25 , find the operating force when the drum rotates in Clock-wise direction and anticlockwise direction. | Apply | CO 4 | PO 2,4 |
| :---: | :---: | :---: | :---: | :---: | :---: |
| 5 | a | Describe the graphical method of determining the inertia of the connecting rod of reciprocating engine. | Understand | CO 4 | PO 1 |
|  | b | The turning moment diagram for a multi cylinder engine has been drawn to a scale of 1 mm to 500 Nm of torque and 1 mm to 60 of crank displacement The intercepted areas between the output torque curve and the mean resistance line taken in order from one end of the engine are $-30,+410,-280,+320,-330,+250,-360,280 \mathrm{~mm}^{2}$ when the engine runs at 800 rpm . The engine has a stroke of 300 mm and the fluctuation of speed is not to exceed $2 \%$ of mean speed. Determine suitable diameter and cross section of the flywheel rim for a limiting value of safe centrifugal stress of 7 Mega Pascal. The material density is $720 \mathrm{~kg} / \mathrm{m}^{3}$. Width of the rim is 5 times the thickness. | Apply | CO 5 | PO 1 |
| 6 | a | Derive an expression for the angular acceleration of the connecting rod of a reciprocating engine. | Understand | CO 4 | PO 2 |
|  | b | The effective turning moment exerted by a two stroke engine at the crank shaft is $T=800+100 \sin 2 \theta-200 \cos \theta$ where $\theta$ is the inclination of the crank to inner dead center. The mass of the flywheel is 400 kg and radius of gyration is 550 mm . The engine speed is 200 rpm . Determine the power developed, the total percentage fluctuation of speed and maximum angular retardation. | Apply | CO 5 | PO 2,4 |
| 7 | a | Explain the balancing of reciprocating masses with a neat sketch. | Understand | CO 7 | PO 1 |


|  | b | A rigid rotor has its unbalance in one plane and can be considered to consist of three masses $m_{1}=5 \mathrm{~kg}$ at an angle of 300 from mass m 1 in anti clockwise direction, $m_{2}=3 \mathrm{~kg}$ at an angle of 1650 counterclockwise from $m_{1}$ and $m_{3}=8 \mathrm{~kg}$ at angle 850 clockwise from $m_{1}$. The radii $r_{1}=200 \mathrm{~mm}, r_{2}=80 \mathrm{~mm}$ and $r_{3}=140 \mathrm{~mm}$. Determine the balancing mass required at a radius of 100 mm . Specify the location of this mass with respect to $m_{1}$. | Apply | CO 7 | PO 2,4 |
| :---: | :---: | :---: | :---: | :---: | :---: |
| 8 | a | Derive the expression for hammer blow in locomotive balancing. | Understand | CO 7 | PO 1 |
|  | b | The cranks of a three cylinder locomotive are set at 1000 . The stroke is 110 mm , the length of the connecting rod is 140 mm , the mass of the reciprocating parts per cylinder is 1 Kg and the speed of the crank shaft is 2400 rpm . Determine the magnitude of primary and secondary balancing. | Apply | CO 7 | PO 2,4 |
| 9 | a | Derive an expression for the natural frequency of forced longitudinal vibrations | Understand | CO 8 | PO 1 |
|  | b | A shaft 100 mm diameter and 1000 mm long is fixed at one end and the other end carries a flywheel of mass 90 kg . The radius of gyration of the flywheel is 500 mm . Find the frequency of torsional vibration, if the modulus of rigidity for the shaft material is $80 \mathrm{GN} / \mathrm{m}^{2}$. | Apply | CO 8 | PO 1 |
| 10 | a | What is meant by the term critical damping? | Understand | CO 9 | PO 1 |
|  | b | A cantilever shaft of 50 mm diameter and 300 mm long has a disc of mass 100 kg at its free end. The Youngs modulus of the shaft material is $200 \mathrm{GN} / \mathrm{m}^{2}$. Determine the frequency of longitudinal and transverse vibrations of the shaft. | Apply | CO 8 | PO 2 |

KNOWLEDGE COMPETENCY LEVELS OF MODEL QUESTION PAPER


Signature of Course Coordinator
HOD, ME
Mr. A Somaiah, Assistant Professor

